

INDIAN MARITIME UNIVERSITY
(A CENTRAL UNIVERSITY, GOVT. OF INDIA)

SEMESTER- II, B.TECH. (MARINE ENGINEERING) – JUNE 2014 EXAMS

APPLIED THERMODYNAMICS - I (T 2203)

(AY 2013-14 batch onwards)

Time:- 3 Hrs
Date: 23.06.2014

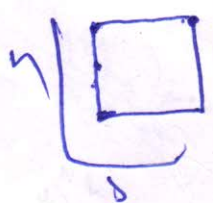
Max Marks : 100
Pass Marks : 50



PART - A
Compulsory Questions

(3 X10 = 30 Marks)

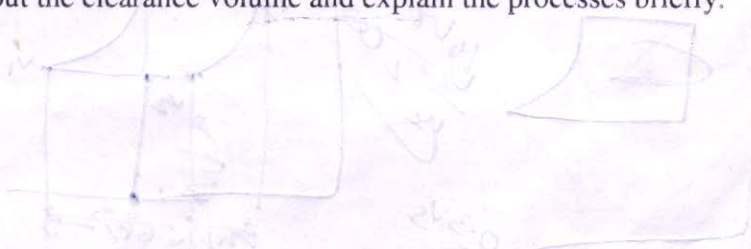
1. a) State Kelvin-Planck statement of second law of Thermodynamics. (3)
- b) What is Entropy? Write down the unit of Entropy in SI system. (3)
- c) Draw the h-s diagram for the Rankine cycle with superheated steam. (3)
- d) What is Mean Effective Pressure (MEP) of a Steam Engine? Write the expression of (MEP) for a steam Engine with clearance volume. (3)
- e) What is volumetric efficiency and clearance ratio of a reciprocating compressor? (3)
- f) What is meant by free air delivered? (3)
- g) Define Indicated power of a steam Engine. (3)
- h) What is multistaging of a compressor? Explain the function of high pressure (HP) cylinder, intercooler, low pressure (LP) cylinder in multistaging. (3)
- i) What are theoretical and actual indicator diagram? (3)
- j) Explain relative humidity. (3)



PART - B
Answer Any Five of the following

(5 X14 = 70 Marks)

2. a) Draw the P-V diagram of a single stage reciprocating compressor without the clearance volume and explain the processes briefly. (7)



- b) A single stage reciprocating compressor takes 1 m^3 of air per minute at 1.013 bar and 15°C and delivers it at 7 bar. Assuming that the law of compression is $PV^{1.35} = C$ and clearance is negligible, calculate the indicated power of the compressor. (7)

Ans - 9.182 kW

- 3) A single-stage single-acting air compressor delivers 0.6 kg of air per minute at 6 bar. The temperature and pressure at the end of suction stroke are 30°C and 1.0 bar. The bore and stroke of the compressor are 100 mm and 150 mm respectively. The clearance volume is 3% of the swept volume. Assume the expansion and compression index to be same and having value 1.3. Find the following: (14)
- i) Volumetric efficiency of the compressor (0.9109)
 - ii) Power required to run the compressor if the Mechanical efficiency is 85%
 - iii) The RPM of the compressor

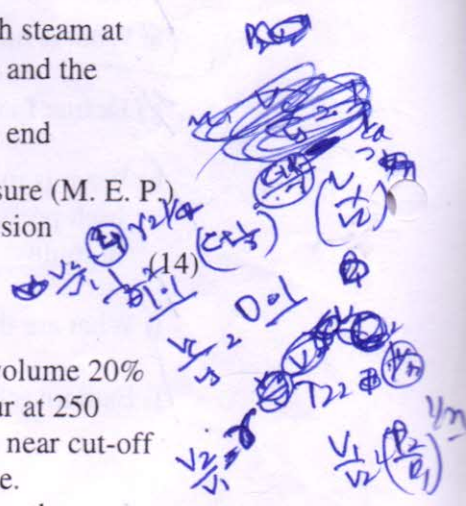
- 4) Air at 103 KPa and 27°C is drawn in the LP cylinder of a two-stage air compressor and is isentropically compressed to 700 KPa. The air is then cooled at constant pressure to 37°C in an intercooler and is then again compressed isentropically to 4 MPa in the high pressure (HP) cylinder and is delivered at this pressure. Determine the power required to run the compressor if it has to deliver 30 m^3 of air per hour measured at inlet condition. (14)

$(\frac{P_2}{P_1})$

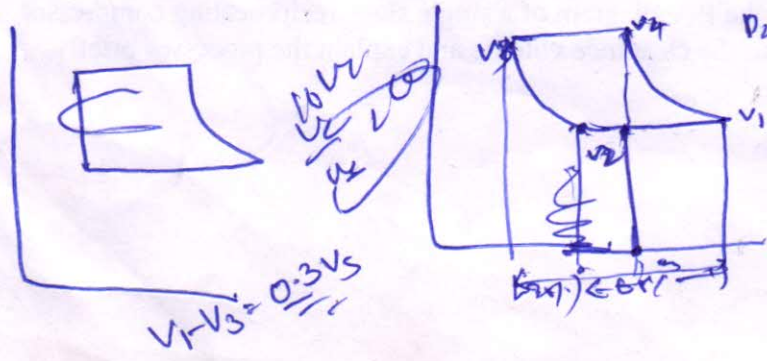
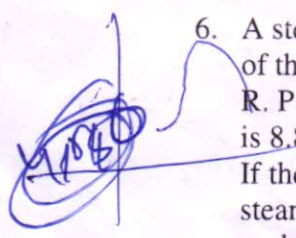
$P_1 V_1 = P_2 V_2$
 $\frac{30}{3600} \times 1000$

$n = \frac{\log \frac{P_2}{P_1}}{\log \frac{V_2}{V_1}}$

- 5) The cylinder of a non-condensing steam engine is supplied with steam at 11.5 bar. The clearance volume is $\frac{1}{10}$ of the stroke volume and the cut-off takes place at $\frac{1}{4}$ of the stroke. If the pressure at the end of the compressor is 5.4 bar, compute the Mean Effective Pressure (M. E. P.) of the steam of piston. Assume that the expansion and compression are hyperbolic. The back pressure is given as 1.1 bar. (14)



6. A steam engine having swept volume 0.034 m^3 and clearance volume 20% of the piston displacement, consumes 2725 kg of steam per hour at 250 R. P. M. The Engine is double acting and the pressure of steam near cut-off is 8.8 bar, when the piston has traversed 30% of working stroke. If the compression commences at 65% of the return stroke when the steam remaining in the cylinder is dry at 0.14 bar, estimate the mass of cushion steam at this point. Also find the dryness fraction of steam at cut-off. (14)



$c = \frac{V_c}{V_s} = \frac{1}{5}$

$V_1 - V_2 = 0.3 V_s$

$D_{22} = 0.0$

$\frac{V_2}{V_3} = 0.0$

$V_{c2} = 0.2 V_s$

$V_{c2} = 0.2 V_s$

